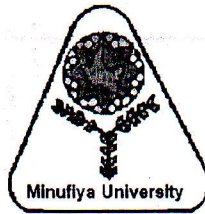


Minoufiya University
Faculty of Engineering
Mechanical Power Eng. Dept
Academic Year: 2015-2016
Date: 11-6-2016



Subject: *Industrial Ventilation*
Code: *MPE 502*
Academic level: *Diploma.*
Time allowed: *3 hours*
Total degree : *100 marks*

Answer all the following questions:

Question-1

[40marks]

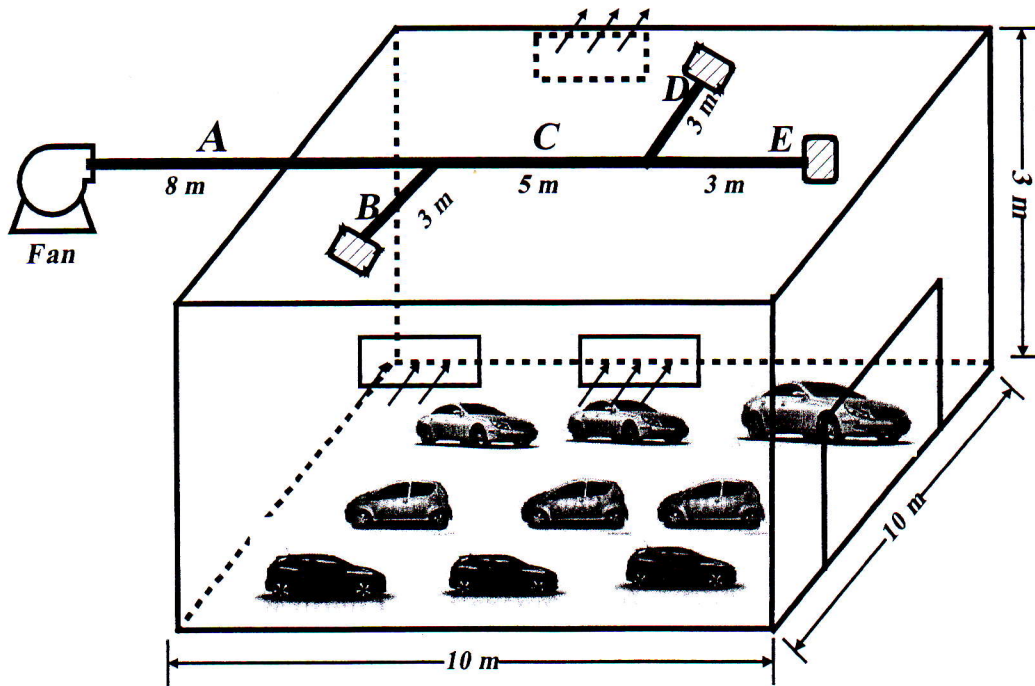
- a- Mention the different sources of pollutants inside the closed spaces. **(6 marks)**
- b- Explain with details the meaning of IAQ. **(6marks)**
- c- Classify the contaminants inside the closed space and mention one example of each type. **(6 marks)**
- d- Describe using diagrammatic sketch the operation of natural ventilation system and report its advantages and disadvantages. **(8 marks)**
- e- Show the difference between the two methods of industrial ventilation system (i.e. dilution and local exhaust), and mention only the disadvantages of each method. **(8 marks)**
- f- What are the general rules for duct design to obtain the optimum performance? **(6 marks)**

Question-2

[60 marks]

- A garage has the dimensions as shown in the next figure. The inlet and exit ventilation openings have the same shape and the same dimensions (25 cm×75 cm). The difference between the levels of inlet and exit opening (i.e. ΔH) is 1.0 m and the discharge coefficient C_D is 0.61 for all the openings. The dynamic pressure head at the inlet opening is neglected. The average temperature inside the garage is 35 °C while the outside temperature is 25 °C. The air flow rate from duct openings are $Q_B=40\%$ of the fan flow rate and $Q_D=Q_E=30\%$ of the fan flow rate. Take the pressure loss coefficients at bend is 0.8 and the exit is 1.0 along the duct. Also, take the velocity for the main duct is 8 m/s and the fan efficiency is 0.82. **Assume any required data and calculate the following:**

- i- The required air flow rate if the air change rate (ACR) equals 6.
- ii- The natural flow rate based on the buoyancy effect only.
- iii- Design the ventilation duct using the equal friction coefficient method.
- iv- The pressure losses at the exit damper.
- v- The fan horse Power (HP).



Use the following relations if you need:

$$\frac{\Delta P_f}{L} = \frac{0.022243 \dot{Q}_{air}^{1.85}}{D^{4.973}}, \quad D_{eq,B} = D_{eq,A} \left(\frac{Q_B}{Q_A} \right)^{\left(\frac{1.85}{4.973} \right)}, \quad D_{eq} = 1.3 \frac{(ab)^{0.625}}{(a+b)^{0.25}}$$

$$\dot{V} = A_e C_e \sqrt{\frac{2g\Delta H\Delta T}{\bar{T}}}, \quad \frac{1}{(A_e C_e)^2} = \frac{1}{(\sum A_i C_i)^2} + \frac{1}{(\sum A_o C_o)^2}$$

With best wishes

Dr. Ashraf Amin